

Optimization of a novel micro-opto-X-ray imaging lens

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Abstract

A mechanically deformable reflection transmission MOEMS system is capable of focusing X rays to sub micron spots. This paper considers the geometry of the proposed deformed slotted microcantilever lens element under thermally derived strain, using finite element analysis. The work shows that an optimized MOEMS design using a slotted polyimide/gold thermal bimorph cantilever is capable of achieving ideal geometry with a larger than expected number of focusing slots up to 111 at 2 μ m width.

Keywords: Micro opto electro mechanical systems; X ray focusing; Microcantilever; Thermal bimorph; Finite element analysis

1. Introduction

Micro-opto-electro-mechanical systems (MOEMS) are of interest for biomedical applications. For instance, studies of the Bystander effect require controlled irradiation of biological cells with focused X-rays to reveal the mechanisms occurring [1,2].

A MOEMS system, designed for this application, is an integration of a flexible microlens, sensor and mechanical actuation mechanism with an electro-mechanical control system to set the require focal length. The system has been given the acronym MOXI, standing for micro-opto-X-ray Imaging. The design and microfabrication of a one dimensional (1-D MOXI) focusing MOEMS system with micro slots for use in X-ray analysis of biological cells was reported in [3]. A 2-D MOXI adaptive optics system for X-ray focusing was suggested in [4]. The present paper focuses on details of the geometry of the deformed slotted micro-beam lens element under thermally derived strain, using finite element analysis, and suggests an optimized MOEMS design, giving prescribed curvature of the lens

through changing the number and the dimensions of the slots.

A 1-D reflective focusing element forming part of a MOXI/MOEMS providing focusing in one direction only is shown in Fig. 1. This is intended for use with a second identical element in cross-coupled Kirkpatrick Baez mode to produce a sub micron spot focus [5].

2. Modeling

The bimorph microlens is a sandwich of layers of polyimide and gold with an embedded metal micro-heater. The difference in linear thermal expansion between the two materials produces a displacement perpendicular to the strain plane. An analytical model assuming cylindrical curvature is inadequate to describe experiments and FEA had to be used. The micro slotted cantilever thermal analysis showed that the temperature propagation is very fast micro seconds and in a controlled area such as a clean room, a homogeneous temperature over the cantilever in the heating and cooling process could be assumed [6]. The chosen software was the SOLID 45 3-D element, which provides for eight nodes with three degrees of dimensional freedom at each node, viz. translations in the x , y ,

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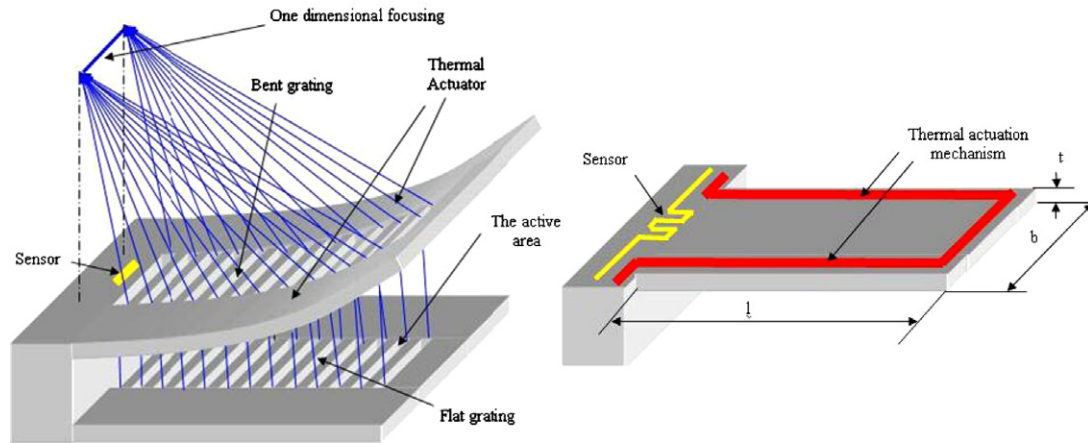


Fig. 1. Focusing action of 1 D MOXI/MOEMS reflection transmission lens.

and z Cartesian directions in the associated ANSYS programme. The dependent variables include creep, swelling, stress, deflection, and strain [7]. In this study, both linear and non-linear analysis were employed and compared with the experimental results from [3]. The model beam consists of polyimide of thickness $7\text{ }\mu\text{m}$ and gold of thickness $1\text{ }\mu\text{m}$. The length of the beam is $450\text{ }\mu\text{m}$ and its width is $120\text{ }\mu\text{m}$. The slotted beam was modeled, assuming it to be initially an arc of a circle fixed at one end and free in the other end, for comparison with the experimental results. The results are shown in Fig. 2 and Table 1.

Both sets of results—those based on a linear FEA and those using a non-linear model—show excellent agreement with experiment at temperatures up to $90\text{ }^{\circ}\text{C}$. However, at $100\text{ }^{\circ}\text{C}$ and above, the model diverges from experiment. The non-linear model shows excellent agreement with experiment up to and including a temperature of $100\text{ }^{\circ}\text{C}$, but this model too shows strong divergence from theory at $100\text{ }^{\circ}\text{C}$. This is consistent with the observations

Table 1

Displacement results using linear and non linear FE analysis compared with experiments

T ($^{\circ}\text{C}$)	Displacement linear FEA (μm)	Displacement non linear FEA (μm)	Experiments (μm)
40	0	0	0
50	5.5	5.83	5
60	11	11	7.5
70	16.49	17.3	17.5
80	21.99	23.6	25
90	27	29.9	32
100	33	36.6	67.5

of Al Aioubi et al. [8], who report irreversible distortions above this temperature due to changes in the thermal and mechanical properties of the polyimide. The thermal expansion coefficient of polyimide increases rapidly—about 40%—above $100\text{ }^{\circ}\text{C}$ [9].

2.1. Optimization

Optimization of the microlens, from the standpoint of its mechanical properties, is an important issue which can be resolved by FEA. The key question is how does the stress in the microcantilever vary with the number of slots? The number of slots should be the maximum possible in order to maximize X-ray transmission, but this is limited by the need to maintain structural integrity under thermal actuation. The modeled dimensions are $450\text{ }\mu\text{m}$ length, $250\text{ }\mu\text{m}$ initial displacement, thickness $8\text{ }\mu\text{m}$. In this optimization process the slot length is fixed; only the number of slots and their width are varied. The proven non-linear FEA approach was used. The data from finite element analysis of slotted beams with 1, 3, 5, 7 and 11 slots are given in Table 2.

The finite element analyses of the slotted beams are compared with a perfect circle. In this analysis the widths of slots are $20\text{ }\mu\text{m}$ and the length $90\text{ }\mu\text{m}$. Another assumption in this section is that, the distance between the slots is equal to the distance of the both end slots to the both free

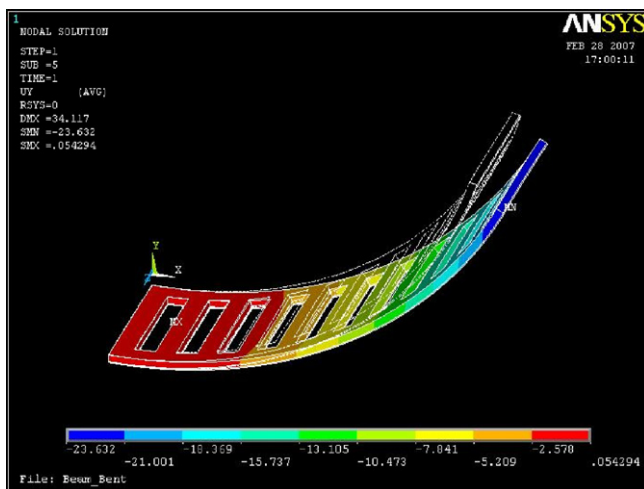


Fig. 2. FEA results showing microcantilever deformation vs. temperature. The microcantilever dimensions are $450 \times 120 \times 7\text{ }\mu\text{m}$; the skeleton shape is the undeformed prestressed bimorph after fabrication with 11 slots.

Table 2

Curvature data obtained from finite element analysis of slotted beams with 1, 3, 5, 7 & 11 slots, compared with perfect circles

Maximum deviation from perfect circle (μ)	Number of slots
1.66	11
2.2	7
2.4	5
2.6	3
2.7	1

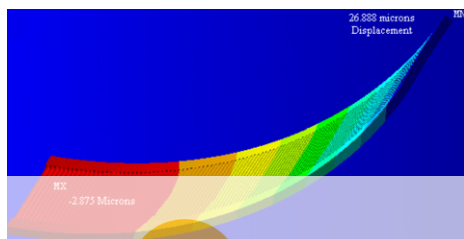


Fig. 3a. FE analysis for an optimized microlens of length 450 μm , 111 slots, 2 μm wide. Minimum displacement is -2.875 μm and the maximum is 26.888 μm .

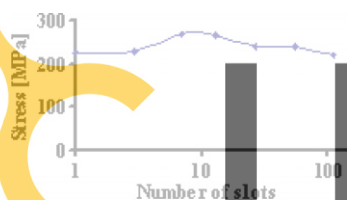


Fig. 3b. Results of FEA optimization showing reduced stress and cylindrical shape for 111 μm wide slots.

Since microfabrication using optical lithography restricts the readily achievable slot width to approximately 2 μm and bearing in mind the foregoing, an optimized structure for our 450 μm long microcantilever would have 111 slots of width 2 μm . The results of non-linear FEA analysis for such a structure are shown in Fig. 3a. The stresses are not strongly dependent on the width of slots, depending rather on their number (Fig. 3b).

3. Conclusions

A series of simulations using FEA analysis has shown the dependence of tip deflection, form and internal stresses on structure and temperature for microcantilever bimorph structures as designed for an X-ray focusing MOEMS (1-D MOXI). The critical importance of the number of slots in obtaining the spherical form necessary for aberration free spot formation has been shown. The superiority of non-linear FEA compared with linear FEA has been revealed by the study, but a simple analytical model is surprisingly good. An optimized structure containing 111 slots 2 μm wide is shown to give a thermally actuated form very close to a perfect circle, as required for the 1-D MOXI MOEMS device.

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and fixed ends of the beam. The analysis has been done for several samples as below, all for the same temperature rise, $\Delta T = 45^\circ\text{C}$.

- 1, 3, 5, 7, 11 slots, 20 μm wide.
- 1, 3, 7, 13, 27 slots, 8 μm wide.
- 1, 3, 7, 13, 27, 55 slots, 4 μm wide.
- 1, 3, 7, 13, 27, 55, 111 slots, 2 μm wide.

From the FEA, there is a clear number of slots for each slot width at which the internal stress in the structure is maximized. Also the curvature, which is arguably the most important parameter in the optimization of the microlens is closer to a perfect circle as the number of slots increases.